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DESIGN AND FINITE ELEMENT ANALYSIS OF TWO WHEELER ENGINE FINS

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ABSTRACT

In India, generally in two wheelers air cooled engines are used. For this purpose, extended surfaces i.e. fins are used, which are mounted on cylinder block and cylinder head. Though the efficiency of cooling in air cooled system is less as compared to the water cooled system still it is used because of less space available to keep accessories. In this project the extended surfaces i.e. fins of Honda Shine & Bajaj Discover two wheeler automobiles are tested to investigate effect on heat transfer rate by changing the Cross-section, Fin Pitch, Fin Material and Fin Thickness. The vehicles considered have single cylinder air cooled engines with set of rectangular fins mounted on the cylinder block. Through experiments temperature generated at steady state condition have been measured from the fin surface and using the value as key parameter, heat dissipated and heat flux through fin is calculated using empirical formulations. The same data is then validated by using FEA approach.

KEYWORDS: Fin, Convection, Heat Transfer.

INTRODUCTION

In case of Internal Combustion engines, combustion of air and fuel takes place inside the engine cylinder and hot gases are generated. The temperature of gases will be around 800 to 1500°C. This is a very high temperature and may result into burning of oil film between the moving parts and may result into seizing or welding of the same. So, this temperature must be reduced to about 150-200°C at which the engine will work most efficiently. Too much cooling is also not desirable since it reduces the thermal efficiency. So, the object of cooling system is to keep the engine running at its most efficient operating temperature. It is to be noted that the engine is quite inefficient when it is cold and hence the cooling system is designed in such a way that it prevents cooling when the engine is warming up and till it attains to maximum efficient operating temperature, then it starts cooling. It is also to be noted that:

- 20-25% of total heat generated is used for producing brake power (useful work).
- Cooling system is designed to remove 30-35% of total heat.

Remaining heat is carried away by exhaust gases. The aim of this project is to find out the effect of fin geometry and fin pitch on cooling of the engine. As the fossil fuel reserves are depleting day by day, the spiraling fuel price is pushing the technology towards its limit to provide engines which are highly efficient and produce high specific power. Air cooled engines are gradually phased out and are being replaced by water cooled engines which are far more efficient in dissipating heat, but in cases of two wheelers and certain other applications, air cooled engines are the only viable option due to space constraints. The heat which is generated during combustion in an internal combustion engine should be maintained at the highest level possible to increase its thermal efficiency, but in order to prevent the thermal damage to the engine components and the lubricants some amount of heat must be removed from the system.

EXPERIMENTAL SETUP



Fig.2.1 : Experimental setup.

Experimental setup is shown in figure 5.1 the setup simply consist of thermocouple rod placed on surface of fin of which temperature readings are to be taken. Thermocouple rod is attached to thermocouple temperature trainer kit which consist of digital display which will provide us actual readings directly. The k-type thermocouple is used in experiment. The readings are taken on stationary engine after reaching to steady state condition. Following observations are found.

| | |
|---------------------------------------|---------------|
| Model name | Honda Shine |
| CC | 125 |
| Stroke(mm) | 58 |
| Bore(mm) | 52 |
| No. of fins | 6 |
| Fin pitch(mm) | 10 |
| Fin thickness(mm) | 2.5 |
| Fin material | Al alloy |
| Position of Fins w.r.t. cylinder axis | Perpendicular |

Table: 2.1.1 Observation for temperature reading of Honda Shine

| Sr. No. | Fin No. | Temperature (°C) |
|---------|---------|------------------|
| 1 | 1 | 129 |
| 2 | 2 | 126 |
| 3 | 3 | 121 |
| 4 | 4 | 120 |

| | | |
|---|---|-----|
| 5 | 5 | 119 |
| 6 | 6 | 115 |

Note: The time to reach steady state was 90 minutes there after the readings were taken.

**Calculations for Peak Temperature Produced in Cylinder of Honda Shine
(Data taken from technical specification of automotive vehicle)**

$$\begin{aligned}
 \text{Initial temperature during suction, } T_1 &= 30 \text{ }^\circ\text{C} \\
 \text{Initial pressure} &P_1 = 1 \text{ bar} \\
 \text{Compression ratio,} &r_c = 9.5 \\
 \text{Peak pressure produced,} &P_3 = 35 \text{ bar} \\
 &P_2 = P_1 \times r_c^\gamma \\
 &= 1 \times (9.5)^{1.4} \\
 &= 23.378 \text{ bar} \\
 T_2 &= T_1 \times (r_c)^{\gamma-1} \\
 &= 303 \times (9.5)^{1.4-1} \\
 &= 745.64 \text{ }^\circ\text{K} \\
 T_3 &= \frac{P_3}{P_2} \times T_2 \\
 &= \frac{35}{23.378} \times 745.64 \\
 &= 1116.32 \text{ K} \\
 &= 843.32 \text{ }^\circ\text{C}
 \end{aligned}$$

Calculation for Heat Dissipated From Surface of Fins of Honda Shine

Assumptions made to calculate the heat dissipated or heat flux:-

- 1) Steady state one dimensional heat conduction.
- 2) Finite long fin and with negligible heat loss from fin tip.
- 3) Constant properties.

Considering, h = Heat transfer coefficient
 A_c = Cross section area of fin
 K = Thermal conductivity
 L = Length of fin.
 w = Width of fin
 t = Thickness of fin
 P = Perimeter of fin
 T_0 = Fin temperature
 T_∞ = Ambient temperature of air.

For fin no.1

$$\begin{aligned}
 A_c &= L \times t = 445 \times 2.5 \times 10^{-6} = 1112.5 \times 10^{-6} \text{ m}^2 \\
 P &= 445 \times 10^{-3} \text{ m} \\
 mL &= \sqrt{\frac{h \times P}{K \times A_c}} \times w = \sqrt{\frac{30 \times 445 \times 10^{-3}}{240 \times 1112.5 \times 10^{-6}}} \times 0.024 = 0.169 \\
 Q_1 &= \sqrt{h \times P \times K \times A_c} \times (T_0 - T_\infty) \times \tanh(mL) \\
 &= \sqrt{30 \times 0.445 \times 240 \times 0.001112} \times (129 - 30) \times \tanh(0.169) \\
 &= \mathbf{31.29 \text{ W.}}
 \end{aligned}$$

Similarly for other fins heat dissipated can be calculated

$$\begin{aligned}
 Q_2 &= 29.52 \text{ W} \\
 Q_3 &= 27.46 \text{ W} \\
 Q_4 &= 26.01 \text{ W} \\
 Q_5 &= 24.65 \text{ W} \\
 Q_6 &= 23.24 \text{ W}
 \end{aligned}$$

Total heat transfer from fins, $Q_{\text{Honda Shine}}=162.17 \text{ W}$

Heat flux Calculation:-

Calculation for fin no.1.

$$\begin{aligned}
 q_1 &= \frac{\text{Heat dissipated by fin}}{\text{Area of cross section for fin}} \\
 &= \frac{Q_1}{A_c} \\
 &= \frac{31.29}{0.00112} \\
 &= 28125.84 \text{ W/m}^2
 \end{aligned}$$

Similarly, heat flux can be calculated for other fins.

$$\begin{aligned}
 q_2 &= 27270.207 \text{ W/m}^2 \\
 q_3 &= 25844.705 \text{ W/m}^2 \\
 q_4 &= 25562.653 \text{ W/m}^2 \\
 q_5 &= 25282.051 \text{ W/m}^2 \\
 q_6 &= 24145.45 \text{ W/m}^2
 \end{aligned}$$

VALIDATION BY USING ANSYS

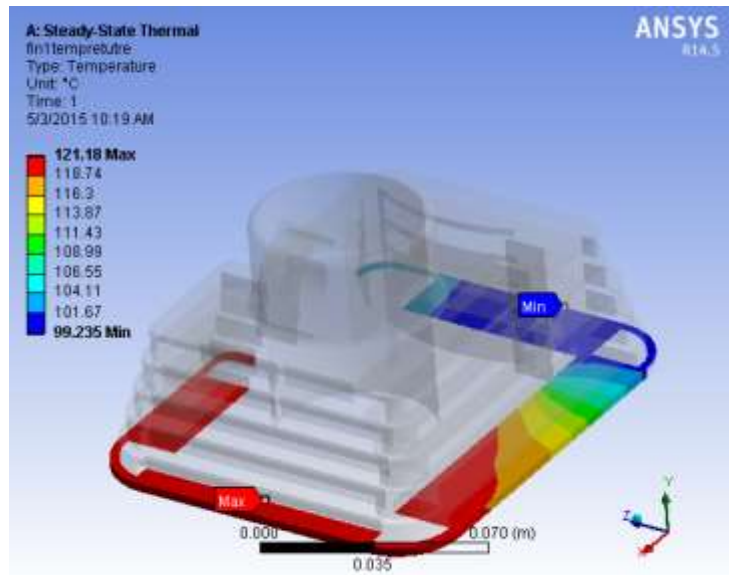


Fig.3.1 : Temperature for Honda Shine Fin 01

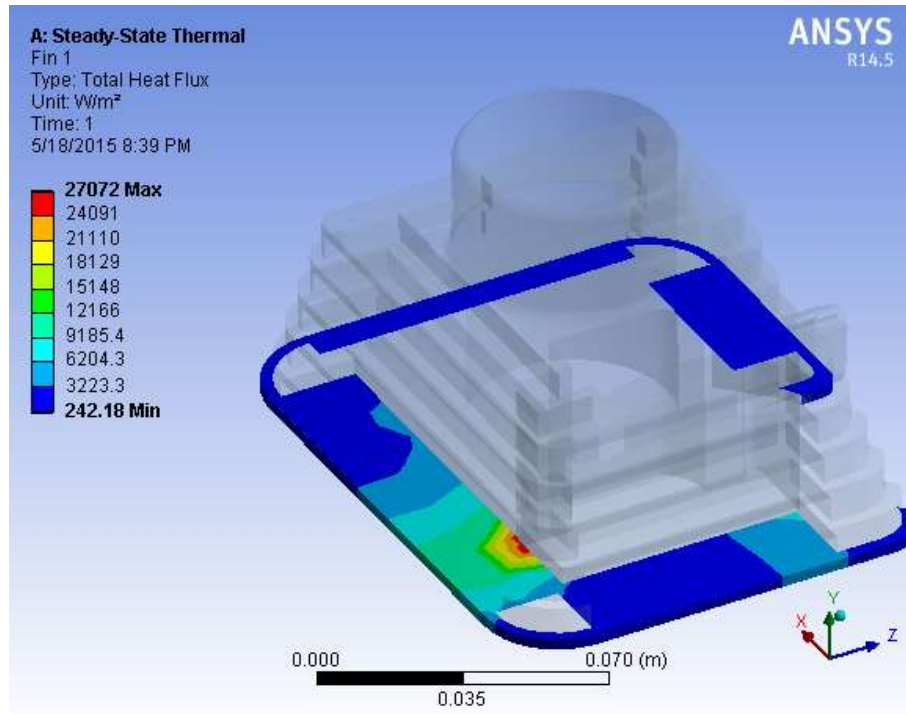


Fig. 3.2.: Heat flux for Honda Shine Fin 01

CONCLUSION

Design of fin plays an important role in heat transfer. The fin geometry and cross sectional area affects the heat transfer coefficient. There is a scope of improvement in heat transfer of air-cooled engine cylinder fin, if fin's shape varied from conventional one. From this project after experiment values and FEA validation it can be concluded that contact surface available for the air to flow over the fin is also important factor in heat transfer rate.

If the turbulence of air is increased by changing the design and geometry (curved and corrugated shaped fins for cylinder block) of the fins, it will increase the rate of heat transfer. Due to non-uniformness in the geometry of fins, turbulence of flowing air increases which results in more heat transfer rate.

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